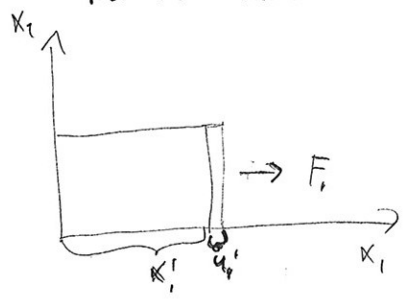


Normal strain and stress



$$\epsilon_1 = \frac{u_1}{x_1'} \quad ? \quad (1)$$

$$\epsilon_1 = \lim_{\Delta x_1 \rightarrow 0} \frac{\Delta u_1}{\Delta x_1} = \frac{\partial u_1}{\partial x_1}$$

$$u_1 = \int_0^{x_1'} \frac{\partial u_1}{\partial x_1} dx_1 = \int_0^{x_1'} \epsilon_1 dx_1$$

Accumulated Average strain

$$\bar{\epsilon}_1 = \int_{x_1'}^{x_1'+u_1} \frac{dx}{x} = \ln x = \ln(x_1'+u_1) - \ln x_1 = \ln \frac{x_1'+u_1}{x_1} = \ln \left(1 + \frac{u_1}{x_1'} \right)$$

$\epsilon_1 \equiv \frac{\partial u_1}{\partial x_1}$
 Normal strain components

So: at constant strain, displacement depends on length.

$$\epsilon_2 = \frac{\partial u_2}{\partial x_2}$$

$$\epsilon_3 = \frac{\partial u_3}{\partial x_3}$$

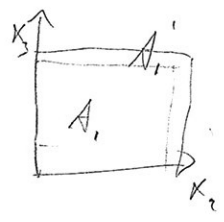
Is there some kind of relation between specimen size and Force needed to induce the strain?

$$\sigma_1 = \frac{\partial F_1}{\partial A_1}$$

$$\sigma_2 = \frac{\partial F_2}{\partial A_2}$$

$$\sigma_3 = \frac{\partial F_3}{\partial A_3}$$

Normal stress components



$$F_{1,2}' = F_{1,2} + \frac{\partial F_{1,2}}{\partial A_1} dA_1$$

$$= F_{1,2} + \sigma_1 (A_1' - A_1)$$

Stress: Internal force

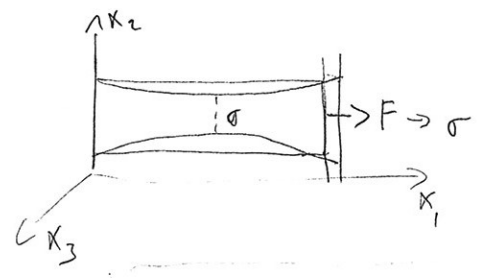
If σ_1 constant:

$$F_1 = \sigma_1 A_1 \Rightarrow \sigma_1 = \frac{F_{1,2}}{A_1}$$

Body experiences Force and Boundary displacement / Measurable
 Material experiences stress and strain
 Material elements

Stress-strain relations - Material behavior (??) (2)

$$\epsilon_1 = \text{function}(\sigma_1) \quad ?$$



$$\epsilon_1 = \text{function}(\sigma_1, \sigma_2, \sigma_3)$$

$$\epsilon_2 = \text{function}(\dots)$$

$$\epsilon_i = \text{function}(\sigma_j), \quad i, j \in \{1, 2, 3\}$$

$$\epsilon_1 = S_{11} \sigma_1 + S_{12} \sigma_2 + S_{13} \sigma_3$$

$$\epsilon_2 = S_{21} \sigma_1 + S_{22} \sigma_2 + S_{23} \sigma_3$$

$$\epsilon_3 = S_{31} \sigma_1 + S_{32} \sigma_2 + S_{33} \sigma_3$$

$$\epsilon_i = \sum_j S_{ij} \sigma_j = S_{ij} \sigma_j$$

$$\boxed{\epsilon_i = S_{ij} \sigma_j}$$

Compliance Matrix

$$\begin{pmatrix} \epsilon_1 \\ \epsilon_2 \\ \epsilon_3 \end{pmatrix} = \begin{pmatrix} S_{11} & S_{12} & S_{13} \\ S_{21} & S_{22} & S_{23} \\ S_{31} & S_{32} & S_{33} \end{pmatrix} \begin{pmatrix} \sigma_1 \\ \sigma_2 \\ \sigma_3 \end{pmatrix}$$

$$\begin{pmatrix} \epsilon \\ \vdots \\ \epsilon_i \end{pmatrix} = \begin{pmatrix} \vdots \\ \vdots \\ S_{ij} \end{pmatrix} \begin{pmatrix} \sigma \\ \vdots \\ \sigma_j \end{pmatrix}$$

$$\sigma_1 = Q_{11} \epsilon_1 + Q_{12} \epsilon_2 + Q_{13} \epsilon_3$$

$$\sigma_2 = Q_{21} \epsilon_1 + \dots$$

$$\sigma_k = Q_{ke} \epsilon_e = \sum_e Q_{ke} \epsilon_e$$

Stiffness matrix

$$\begin{pmatrix} \sigma_1 \\ \sigma_2 \\ \sigma_3 \end{pmatrix} = \begin{pmatrix} Q_{11} & Q_{12} & Q_{13} \\ Q_{21} & Q_{22} & Q_{23} \\ Q_{31} & Q_{32} & Q_{33} \end{pmatrix} \begin{pmatrix} \epsilon_1 \\ \epsilon_2 \\ \epsilon_3 \end{pmatrix}$$

Matrices characteristic for the material
 - linear (idealized)

Young's modulus = modulus of elasticity??

Examples - First Volumetric strain 3.9.06
Uniaxial stress in j-direction: ③

$$\epsilon_i = S_{ij} \sigma_j$$

$$\epsilon_j = S_{jj} \sigma_j \quad (\text{no sum}) = \frac{1}{E_j} \sigma_j$$

$$i \neq j: \epsilon_i = -\nu_{ij} \epsilon_j = -\nu_{ij} S_{jj} \sigma_j \quad (\text{no sum})$$

torsion $\epsilon_i = S_{ij} \sigma_j \quad (\text{no sum})$

$$\Rightarrow S_{ij} = -\nu_{ij} S_{jj} = -\frac{\nu_{ij}}{E_j}$$

~~Homogeneous~~ Isotropic Incompressible material:
 Uniaxial stress in j

$$\sum_i \epsilon_i = 0 = \epsilon_j + \epsilon_k + \epsilon_l = 0 \quad j \neq k \neq l$$

$$\epsilon_j = S_{jj} \sigma_j$$

$$\epsilon_i + \epsilon_k + \epsilon_l = 0 = \text{Volumetric strain} = 1 - (1 + \epsilon_1)(1 + \epsilon_2)(1 + \epsilon_3)$$

$$\epsilon_k = \dots$$

$$\epsilon_j + \epsilon_k + \epsilon_l =$$

$$\epsilon_k = -\nu_{kj} S_{jj} \sigma_j$$

$$(1 - \nu - \nu) S_{jj} \sigma_j = 0$$

$$\epsilon_l = -\nu_{lj} S_{jj} \sigma_j$$

$$\Rightarrow \nu = \frac{1}{2}$$

$$\nu_{kj} = \nu_{lj} = \nu \quad (\text{Isotropy})$$

Volumetric Strain

$$\epsilon_v = (1 + \epsilon_1)(1 + \epsilon_2)(1 + \epsilon_3) - 1 \approx \epsilon_1 + \epsilon_2 + \epsilon_3$$

Total differential of displacement

$$du_i = \frac{\partial u_i}{\partial x_j} dx_j = \sum_j \epsilon_{ij} dx_j = \epsilon_{ij} dx_j$$

$$du_1 = \epsilon_{11} dx_1 + \epsilon_{12} dx_2 + \epsilon_{13} dx_3$$

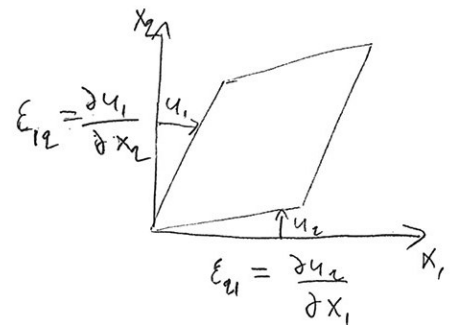
$$du_2 = \epsilon_{21} dx_1 + \epsilon_{22} dx_2 + \epsilon_{23} dx_3$$

$$du_3 = \epsilon_{31} dx_1 + \epsilon_{32} dx_2 + \epsilon_{33} dx_3$$

$$\epsilon_{ij} = \frac{\partial u_i}{\partial x_j}$$

What are these $\epsilon_{ij}, i \neq j$?

Shear strains



$$\begin{pmatrix} du_1 \\ du_2 \\ du_3 \end{pmatrix} = \begin{pmatrix} \epsilon_{11} & \epsilon_{12} & \epsilon_{13} \\ \epsilon_{21} & \epsilon_{22} & \epsilon_{23} \\ \epsilon_{31} & \epsilon_{32} & \epsilon_{33} \end{pmatrix} \begin{pmatrix} dx_1 \\ dx_2 \\ dx_3 \end{pmatrix}$$

Strain tensor, (symmetric)

→ 6 indep. components

Total Differential of Force

3.9.06
⑤

$$dF_i = \frac{\partial F_i}{\partial A_j} dA_j = \sum_j \sigma_{ij} dA_j = \sigma_{ij} dA_j$$

$$\begin{pmatrix} dF_1 \\ dF_2 \\ dF_3 \end{pmatrix} = \begin{pmatrix} \sigma_{11} & \sigma_{12} & \sigma_{13} \\ \sigma_{21} & \sigma_{22} & \sigma_{23} \\ \sigma_{31} & \sigma_{32} & \sigma_{33} \end{pmatrix} \begin{pmatrix} dA_1 \\ dA_2 \\ dA_3 \end{pmatrix}$$

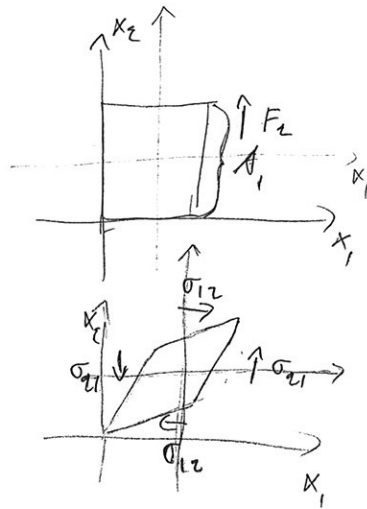
Stress tensor

Symmetric \rightarrow 6 indep. comp.

Shear stresses

$$\sigma_{ij}, i \neq j$$

$$\sigma_{ij} = \frac{\partial F_i}{\partial A_j}$$

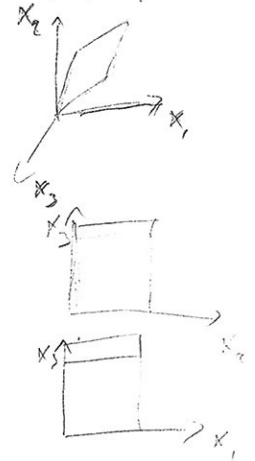


ON-AXIS Stress-strain relations for orthogonal axis of ⑥

3.9.06

Symmetry = **ORTHOTROPY**

	ϵ_{11}	ϵ_{22}	ϵ_{33}	$2\epsilon_{12}$	$2\epsilon_{13}$	$2\epsilon_{23}$
ϵ_{11}	S_{1111}	S_{1122}	S_{1133}			
ϵ_{22}	S_{2211}	S_{2222}	S_{2233}			
ϵ_{33}	S_{3311}	S_{3322}	S_{3333}			
ϵ_{12}				S_{1212}		
ϵ_{13}					S_{1313}	
ϵ_{23}						S_{2323}
	Symmetric					

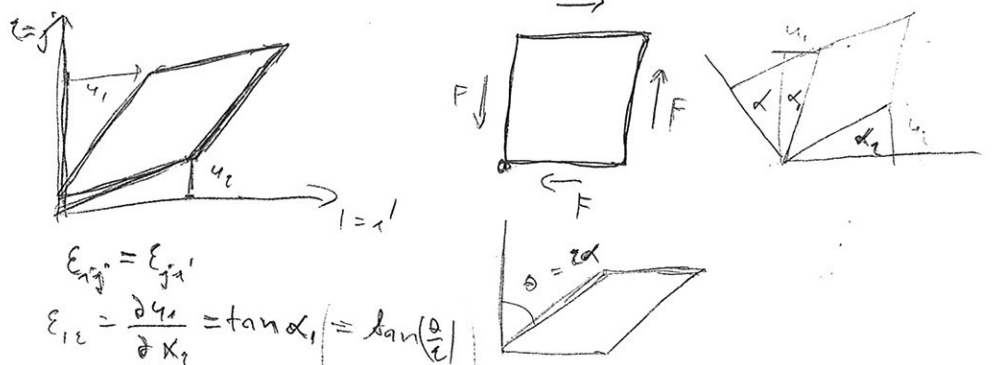


	σ_{11}	σ_{22}	σ_{33}	$2\sigma_{12}$	$2\sigma_{13}$	$2\sigma_{23}$
σ_{11}	Q_{1111}	Q_{1122}	Q_{1133}			
σ_{22}	Q_{2211}	Q_{2222}	Q_{2233}			
σ_{33}	Q_{3311}	Q_{3322}	Q_{3333}			
σ_{12}				Q_{1212}		
σ_{13}					Q_{1313}	
σ_{23}						Q_{2323}
	Symmetric					

Measurement of shear strain

11.9.06

(7)



$$\epsilon_{11} = \epsilon_{22}$$

$$\epsilon_{12} = \frac{\partial u_1}{\partial x_2} = \tan \alpha_1 = \tan\left(\frac{\theta}{2}\right)$$

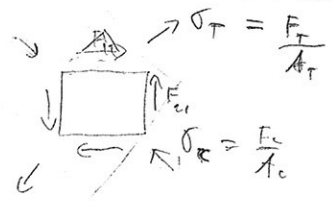
$$\epsilon_{21} = \frac{\partial u_2}{\partial x_1} = \tan \alpha_2 = \tan\left(\frac{\theta}{2}\right)$$

$$\epsilon_{11} + \epsilon_{22} = 2\epsilon_{12} = \tan \alpha_1 + \tan \alpha_2$$

$$= 2 \tan \alpha$$

$$= 2 \tan\left(\frac{\theta}{2}\right)$$

Measurement of shear stress



$$\sigma_t = \frac{F_t}{A_t} \Rightarrow F_t = \sigma_t A_t$$

$$F_t = \cos 45^\circ F_{12} + \sin 45^\circ F_{21} = \frac{2}{\sqrt{2}} F_{12} = \sqrt{2} F_{12}$$

$$\sigma_{12} = \frac{F_{12}}{A_{12}} = \frac{F_{12}}{2 \cos 45^\circ \frac{A_t}{2}} = \frac{F_{12}}{\frac{1}{\sqrt{2}} A_t} = \frac{F_t}{\sqrt{2} A_t} = \frac{F_t}{A_t} = \sigma_t$$

$$\sigma_{12} + \sigma_{21} = \sigma_t$$

$$\sigma_c = \sin 45^\circ F_{12} + \cos 45^\circ F_{21} = \sqrt{2} F_{12}$$

$$\sigma_{12} = \frac{F_{12}}{A_{12}} = \frac{F_{12}}{2 \cos 45^\circ \frac{A_c}{2}} = \frac{F_{12}}{\frac{1}{\sqrt{2}} A_c} = \frac{F_c}{\sqrt{2} A_c} = \frac{F_c}{A_c} = \sigma_c$$

Tulo objekti sama
suurennus jännityksen
muunnokseen

$$\begin{pmatrix} m^1 & n^1 & 2mn \\ n^1 & m^1 & -2mn \\ -mn & mn & m^2 - n^2 \end{pmatrix}$$

Shear compliance

$$S_{ijij} = \frac{\epsilon_{ij}}{2\sigma_{ij}} = \frac{\tan \alpha}{\sigma_t} = \frac{\tan \frac{\theta}{2}}{\sigma_c}$$

shear stiffness

$$Q_{ijij} = \frac{\sigma_{ij}}{2\epsilon_{ij}} = \frac{1}{4} \frac{1}{S_{ijij}}$$

Uniaxial stress experiments

in j

4.9.06

(8)

$$\epsilon_{ii} = S_{ijij} \sigma_{jj} \quad (\text{no sum})$$

$$\sigma_{jj} = E_j \epsilon_{jj} \Rightarrow S_{ijij} = \frac{1}{E_j}$$

$i \neq j$

$$\epsilon_{ii} = -\nu_{ij} \epsilon_{jj} = -\frac{\nu_{ij}}{E_j} \sigma_{jj} \Rightarrow S_{iijj} = -\frac{\nu_{ij}}{E_j}$$

Pure Shear Experiments

$$\epsilon_{ij} = S_{ijij} 2\sigma_{ij}$$

$$\sigma_{ij} = Q_{ijij} 2\epsilon_{ij} = Q_{ijij} 2\epsilon_{ij}$$

$$\Rightarrow S_{ijij} = \frac{1}{4Q_{ijij}} = \frac{1}{4} \frac{1}{Q_{ijij}}$$

Engineering constants

$$E_j = \frac{1}{S_{jjjj}}$$

$$\nu_{ij} = -\frac{S_{iijj}}{S_{jjjj}} = -S_{iijj} E_j$$

$$S_{ij} = \frac{1}{4S_{ijij}}$$

symmetry of compliance matrix:

$$S_{ijij} = S_{jjii}$$

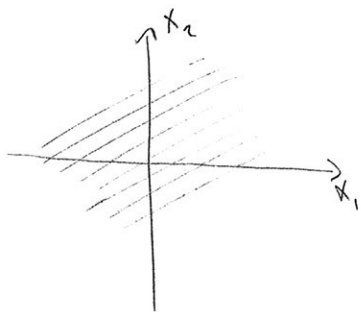
$$\Rightarrow \frac{\nu_{ij}}{E_j} = \frac{\nu_{ji}}{E_i}$$

ON-Axis
ORTHOTROPIC RELATIONS

OFF-AXIS mechanical behavior

6.9.06

(9)



(Determination of compliance:)

(OFF-axis stresses known
→ determine off-axis strains)

- 1° Stress transformation to on-axis
- 2° Use on-axis compliance matrix
→ on-axis strains
- 3° Negative transformation of strain
to off-axis

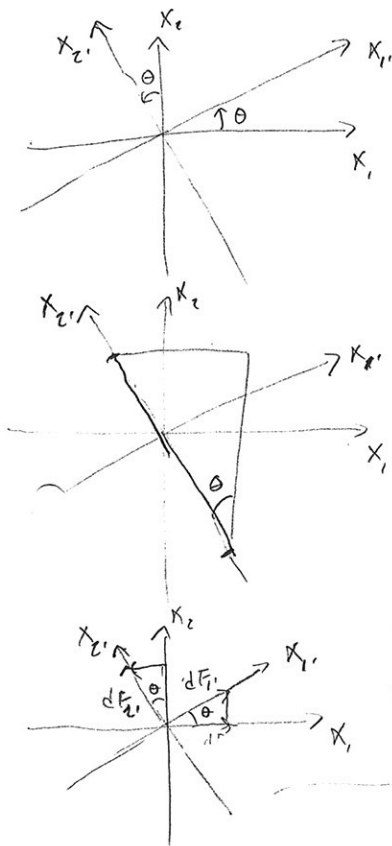
Determine off-axis stress
(strains known)

- 1° Strain transformation to on-axis
- 2° Use stiffness matrix
- 3° Negative transformation of stress

Co-ordinate transformation of stress

5.9.06

(10)



$$dF_i = \sum_j \frac{\partial F_i}{\partial A_j} dA_j$$

$$\begin{aligned} dF_1 &= \frac{\partial F_1}{\partial A_1} dA_1 + \frac{\partial F_1}{\partial A_2} dA_2 \\ &= \sigma_{11} d(\cos\theta A_{1'}) + \sigma_{12} d(\sin\theta A_{1'}) \\ &= \sigma_{11} \cos\theta dA_{1'} + \sigma_{12} \sin\theta dA_{1'} \end{aligned}$$

$$\begin{aligned} dF_1 &= \cos\theta dF_{1'} - \sin\theta dF_{2'} \\ &= \cos\theta [\sigma_{1'1'} dA_{1'} + \sigma_{1'2'} dA_{2'}] - \sin\theta [\sigma_{2'1'} dA_{1'} + \sigma_{2'2'} dA_{2'}] \end{aligned}$$

$$\frac{dF_1}{dA_{1'}} = \cos\theta \sigma_{1'1'} + \sin\theta \sigma_{1'2'}$$

$$\frac{dF_1}{dA_{1'}} = \cos\theta \sigma_{1'1'} - \sin\theta \sigma_{2'1'}$$

$$dF_2 = \sigma_{21} dA_1 + \sigma_{22} dA_2 = \cos\theta \sigma_{21} dA_{1'} + \sin\theta \sigma_{22} dA_{1'}$$

$$\begin{aligned} dF_2 &= \sin\theta dF_{1'} + \cos\theta dF_{2'} \\ &= \sin\theta [\sigma_{1'1'} dA_{1'} + \sigma_{1'2'} dA_{2'}] + \cos\theta [\sigma_{2'2'} dA_{2'} + \sigma_{2'1'} dA_{1'}] \end{aligned}$$

$$\frac{dF_2}{dA_{1'}} = \cos\theta \sigma_{21} + \sin\theta \sigma_{22}$$

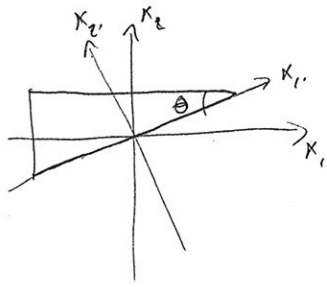
$$\frac{dF_2}{dA_{1'}} = \sin\theta \sigma_{1'1'} + \cos\theta \sigma_{2'1'}$$

$$\begin{cases} m \sigma_{11} + n \sigma_{12} = m \sigma_{1'1'} + n \sigma_{1'2'} \\ n \sigma_{22} + m \sigma_{12} = n \sigma_{1'1'} + m \sigma_{1'2'} \end{cases}$$

Solve
⇒ $\sigma_{1'1'} = \dots$
 $\sigma_{1'2'} = \dots$

Cr'd transform of stress contd

5.9.06
(11)



$$dF_1 = \sigma_{11} dA_1 + \sigma_{12} dA_2 = \sigma_{11} \sin\theta dA_2' + \sigma_{12} \cos\theta dA_2'$$

$$\frac{dF_1}{dA_2'} = \sigma_{11} \sin\theta + \sigma_{12} \cos\theta$$

$$\frac{dF_1}{dA_2'} = \cos\theta \sigma_{1'2'} - \sin\theta \sigma_{2'2'}$$

$$n \sigma_{11} + m \sigma_{12} = m \sigma_{1'2'} - n \sigma_{2'2'}$$

$$\Rightarrow \sigma_{2'2'} = \frac{m}{n} (\sigma_{1'2'} - \sigma_{12}) - \sigma_{11} \Rightarrow \left. \begin{matrix} \sigma_{1'1'} \\ \sigma_{2'2'} \\ \sigma_{1'2'} \end{matrix} \right\} \text{All known:}$$

	σ_{11}	σ_{22}	σ_{12}
$\sigma_{1'1'}$	m^2	n^2	$2mn$
$\sigma_{2'2'}$	n^2	m^2	$-2mn$
$\sigma_{1'2'}$	$-mn$	mn	$m^2 - n^2$

Invariant: $\sigma_{1'1'} + \sigma_{2'2'} = (m^2 + n^2)(\sigma_{11} + \sigma_{22}) = \sigma_{11} + \sigma_{22}$

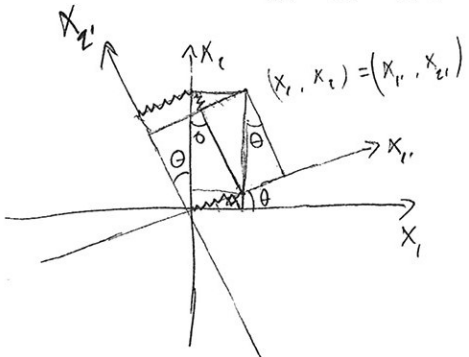
check Inversibility of stress transformation 6.9.06
(12)

	$m^2 \sigma_{11} + n^2 \sigma_{22} - 2mn \sigma_{12}$	$n^2 \sigma_{11} + m^2 \sigma_{22} + 2mn \sigma_{12}$	$mn \sigma_{11} - mn \sigma_{22} + (m^2 - n^2) \sigma_{12}$
σ_{11}	m^2	n^2	$2mn$
σ_{22}	n^2	m^2	$-2mn$
σ_{12}	$-mn$	mn	$m^2 - n^2$

$$\sigma_{11} = m^4 \sigma_{11} + m^2 n^2 \sigma_{22} - 2m^3 n \sigma_{12} + n^4 \sigma_{11} + m^2 n^2 \sigma_{22} + 2m n^3 \sigma_{12} + 2m^3 n^2 \sigma_{11} - 2m^2 n^2 \sigma_{22} + 2(m^3 n - m n^3) \sigma_{12} = (m^2 + n^2)^2 \sigma_{11} = \sigma_{11}$$

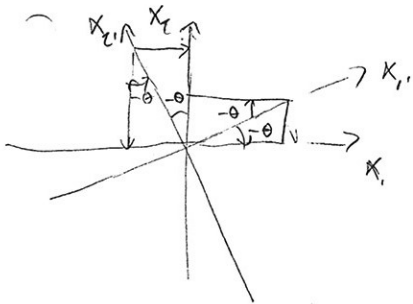
$$\begin{pmatrix} \sigma_{11} \\ \sigma_{22} \\ \sigma_{12} \end{pmatrix} = (R) (R^{-1}) \begin{pmatrix} \sigma_{11} \\ \sigma_{22} \\ \sigma_{12} \end{pmatrix}$$

$$\begin{pmatrix} m^2 & n^2 & 2mn \\ n^2 & m^2 & -2mn \\ -mn & mn & m^2 - n^2 \end{pmatrix} \begin{pmatrix} m^2 & n^2 & -2mn \\ n^2 & m^2 & 2mn \\ mn & -mn & m^2 - n^2 \end{pmatrix} = \begin{pmatrix} m^4 + m^2 n^2 + 2m^2 n^2 & m^2 n^2 + m^2 n^2 - 2m^2 n^2 & -2m^3 n^2 - 2m^2 n^2 + 2m^2 n^2 \\ n^2 m^2 + m^2 n^2 - 2m^2 n^2 & n^2 m^2 + m^2 n^2 - 2m^2 n^2 & -2m^2 n^2 + 2m^2 n^2 \\ -m^3 n + m^2 n^2 + m^2 n^2 - m^2 n^2 & -m^2 n^2 + m^2 n^2 & -2m^2 n^2 + 2m^2 n^2 \end{pmatrix} = \begin{pmatrix} 1 & 0 & 0 \\ 0 & 1 & 0 \\ 0 & 0 & 1 \end{pmatrix}$$



$$X_1' = \cos\theta X_1 + \sin\theta X_2$$

$$X_2' = \cos\theta X_2 - \sin\theta X_1$$



$$X_1 = \cos\theta X_1' + \sin(-\theta) X_2'$$

$$= \cos\theta X_1' - \sin\theta X_2'$$

$$X_2 = \cos\theta X_2' + \sin\theta X_1'$$

$$= \sin\theta X_1' + \cos\theta X_2'$$

$$u_1' = m u_1 + n u_2$$

$$u_2' = -n u_1 + m u_2$$

$$u_1 = m u_1' - n u_2'$$

$$u_2 = n u_1' + m u_2'$$

$$E_{1'2'} = \frac{\partial u_{1'}}{\partial X_{2'}} = \frac{\partial u_1}{\partial X_1} \frac{\partial X_1}{\partial X_{2'}} + \frac{\partial u_1}{\partial X_2} \frac{\partial X_2}{\partial X_{2'}}$$

$$= \frac{\partial u_1}{\partial X_1} (-n) + \frac{\partial u_1}{\partial X_2} m$$

$$= \left[m \frac{\partial u_1}{\partial X_1} + n \frac{\partial u_2}{\partial X_1} \right] (-n) + \left[m \frac{\partial u_1}{\partial X_2} + n \frac{\partial u_2}{\partial X_2} \right] m$$

$$= -mn E_{11} + (m^2 - n^2) E_{12} + mn E_{22}$$

$$E_{1'j'} = \frac{\partial u_{i'}}{\partial X_{j'}} \quad E_{i'j'} = \frac{\partial u_{i'}}{\partial X_{j'}}$$

$$E_{1'j'} = \frac{\partial u_{i'}}{\partial X_{j'}} \frac{\partial X_{j'}}{\partial X_{j'}} \quad \text{sum!}$$

$$E_{1'1'} = \frac{\partial u_{1'}}{\partial X_1} \frac{\partial X_1}{\partial X_{1'}} + \frac{\partial u_{1'}}{\partial X_2} \frac{\partial X_2}{\partial X_{1'}}$$

$$= \frac{\partial u_{1'}}{\partial X_1} m + \frac{\partial u_{1'}}{\partial X_2} n$$

$$= \left[m \frac{\partial u_1}{\partial X_1} + n \frac{\partial u_2}{\partial X_1} \right] m + \left[m \frac{\partial u_1}{\partial X_2} + n \frac{\partial u_2}{\partial X_2} \right] n$$

$$= m^2 E_{11} + 2nm E_{12} + n^2 E_{22}$$

$$E_{2'2'} = \frac{\partial u_{2'}}{\partial X_{2'}} = \frac{\partial u_2}{\partial X_1} \frac{\partial X_1}{\partial X_{2'}} + \frac{\partial u_2}{\partial X_2} \frac{\partial X_2}{\partial X_{2'}}$$

$$= \frac{\partial u_2}{\partial X_1} (-n) + \frac{\partial u_2}{\partial X_2} m$$

$$= \left[-n \frac{\partial u_1}{\partial X_1} + m \frac{\partial u_2}{\partial X_1} \right] (-n) + \left[-n \frac{\partial u_1}{\partial X_2} + m \frac{\partial u_2}{\partial X_2} \right] m$$

$$= n^2 E_{11} - 2nm E_{12} + m^2 E_{22}$$

$$\begin{pmatrix} m^2 & n^2 & 2nm \\ n^2 & m^2 & -2nm \\ -mn & mn & m^2 - n^2 \end{pmatrix} \begin{pmatrix} m^2 & n^2 & -2nm \\ n^2 & m^2 & 2nm \\ mn & -mn & m^2 - n^2 \end{pmatrix} =$$

$$\begin{pmatrix} m^4 + n^4 + 2m^2n^2 & m^2n^2 + m^2n^2 - 2m^2n^2 & -2m^2n + 2n^2m + 2m^2n - 2n^2m \\ n^2m^2 + n^2m^2 - 2n^2m^2 & n^4 + m^4 + 2m^2n^2 & -2n^2m + 2m^2n - 2m^2n + 2n^2m \\ -m^2n - n^2m + m^2n - n^2m & -n^2m + m^2n - m^2n + n^2m & 2m^2n^2 + 2m^2n^2 + m^4 + n^4 - 2m^2n^2 \end{pmatrix}$$

$$= \begin{pmatrix} 1 & 0 & 0 \\ 0 & 1 & 0 \\ 0 & 0 & 1 \end{pmatrix} \%$$

Co-ordinate transformations of compliance and stiffness

6.9.06
15

ON-AXIS → OFF-AXIS

	S_{1111}	S_{1122}	S_{3333}	S_{1122}	S_{1133}	S_{2233}	S_{1212}	S_{1313}	S_{2323}
S'_{1111}	<input type="checkbox"/>	<input type="checkbox"/>		<input type="checkbox"/>			<input type="checkbox"/>		
S'_{1122}	<input type="checkbox"/>	<input type="checkbox"/>		<input type="checkbox"/>			<input type="checkbox"/>		
S'_{3333}			<input type="checkbox"/>						
S'_{1122}	<input type="checkbox"/>	<input type="checkbox"/>		<input type="checkbox"/>			<input type="checkbox"/>		
S'_{1133}					<input type="checkbox"/>				
S'_{2233}						<input type="checkbox"/>			
S'_{1212}	<input type="checkbox"/>	<input type="checkbox"/>		<input type="checkbox"/>			<input type="checkbox"/>		
S'_{1313}								<input type="checkbox"/>	
S'_{2323}									<input type="checkbox"/>
S'_{1112}	<input type="checkbox"/>	<input type="checkbox"/>		<input type="checkbox"/>			<input type="checkbox"/>		
S'_{1113}					<input type="checkbox"/>				
S'_{2212}	<input type="checkbox"/>	<input type="checkbox"/>		<input type="checkbox"/>			<input type="checkbox"/>		
S'_{2223}									<input type="checkbox"/>
S'_{3313}									
S'_{3323}									
S'_{1213}									
S'_{1223}									
S'_{1323}									

- 1° Transform stress to on-axes
- 2° Use on-axis compliances to compute on-axis strains
- 3° Transform strain to off-axes
- 4° Identify off-axis compliance matrix elements
- 5° Identify coefficients of any element w.r.t. any on-axis element

time-dependent behavior

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$$\epsilon_{ij} = \text{function}(\sigma_{ke})$$

$$\epsilon_{ij} = S_{ijkl} \sigma_{ke} = \sum_k \sum_l S_{ijkl} \sigma_{ke}$$

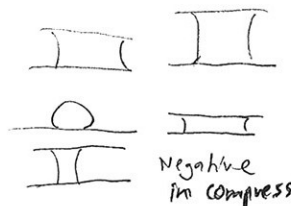
$$= S_{ij11} \sigma_{11} + S_{ij22} \sigma_{22} + S_{ij33} \sigma_{33} + \underbrace{2 S_{ij12} \sigma_{12} + 2 S_{ij13} \sigma_{13} + 2 S_{ij23} \sigma_{23}}_{S_{ij12} \sigma_{12} + S_{ij21} \sigma_{21}}$$

$$\sigma_{ij} = \text{function}(\epsilon_{ke})$$

$$\sigma_{ij} = Q_{ijkl} \epsilon_{ke}$$

Is this right?

What is the Young's modulus $\frac{1}{S_{11}}$ of water?



$$\sigma_{ij} = \text{function}(\dot{\epsilon}_{ke}) = \text{function}\left(\frac{d\epsilon_{ke}}{dt}\right)$$

Newtonian viscosity: $\sigma_{ij} = \eta_{ijkl} \dot{\epsilon}_{ke} \delta_{ik} \delta_{jl}$, $i \neq j$

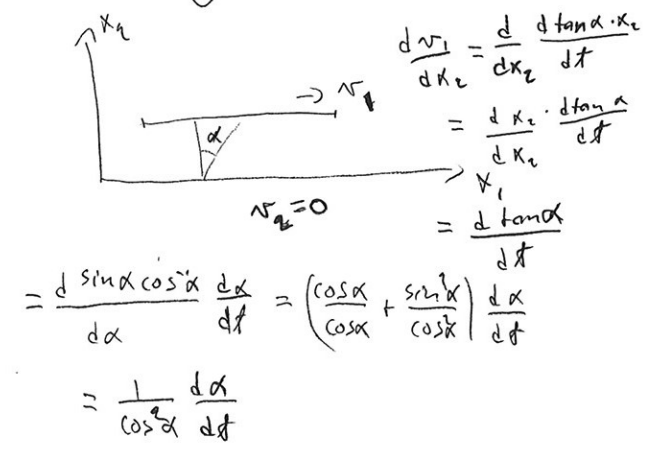
Isotropic fluid: $\sigma_{ij} = \eta \frac{d\epsilon_{kk}}{dt} \delta_{ij}$, $i \neq j \rightarrow \mu \frac{d \tan \alpha}{dt}$

$$\sigma_{ij} = \frac{\partial F_i}{\partial x_j}$$

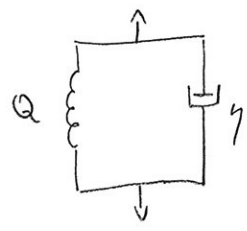
$$\epsilon_{kk} = \frac{\partial u_k}{\partial x_k}$$

Generally: $\sigma_{ij} = Q_{ijkl} \epsilon_{ke} + \eta_{ijkl} \dot{\epsilon}_{ke} \delta_{ij} = \frac{1}{\cos^2 \alpha} \frac{d\alpha}{dt} \approx \frac{d\alpha}{dt}$

How about ϵ_{ij} ?



The Kelvin model



$$\sigma = Q\varepsilon + \eta \dot{\varepsilon}$$

$$\frac{\varepsilon}{\sigma} = \frac{1}{Q} - \frac{\eta}{\sigma Q} \frac{d\varepsilon}{dt}$$

$$\frac{\varepsilon}{\sigma} - \frac{1}{Q} = -\frac{\tau}{\sigma} \frac{d\varepsilon}{dt}$$

$$\frac{1}{\frac{\varepsilon}{\sigma} - \frac{1}{Q}} = -\frac{\sigma}{\tau} \frac{dt}{d\varepsilon}$$

$$\frac{d\varepsilon}{\varepsilon - \frac{\sigma}{Q}} = -\frac{dt}{\tau}$$

Creep experiment:

$\sigma = \text{constant}$

$$\frac{d \ln(\varepsilon - \frac{\sigma}{Q})}{d\varepsilon} = \frac{d \ln(\quad)}{d(\varepsilon - \frac{\sigma}{Q})} \frac{d(\varepsilon - \frac{\sigma}{Q})}{d\varepsilon}$$

$$= \frac{1}{\varepsilon - \frac{\sigma}{Q}}$$

$$\frac{d\varepsilon}{\varepsilon - \frac{\sigma}{Q}} = -\frac{dt}{\tau} \quad | \int$$

$$\ln\left(\frac{\varepsilon}{\sigma} - \frac{1}{Q}\right) = -\frac{t}{\tau} + C$$

$$\varepsilon - \frac{\sigma}{Q} = c_2 e^{-\frac{t}{\tau}}$$

$$\varepsilon = \frac{\sigma}{Q} + c_2 e^{-\frac{t}{\tau}}$$

$$= \varepsilon_\infty + (\varepsilon_0 - \varepsilon_\infty) e^{-\frac{t}{\tau}}$$

$$= \varepsilon_0 e^{-\frac{t}{\tau}} + \varepsilon_\infty (1 - e^{-\frac{t}{\tau}})$$

$$= \varepsilon_\infty (1 - e^{-\frac{t}{\tau}})$$

$$\varepsilon = \sigma \int_{\infty} (1 - e^{-\frac{t}{\tau}})$$

$\tau \equiv$ Retardation time

The Maxwell model



$$\varepsilon = \varepsilon^e + \varepsilon^i$$

$$\sigma = Q\varepsilon^e$$

$$\sigma = \eta \dot{\varepsilon}^i$$

$$\dot{\varepsilon}^i = \frac{d\varepsilon^i}{dt} = \frac{Q\varepsilon^e}{\eta} = \frac{\sigma}{\eta}$$

Steady creep, if σ constant

$$\varepsilon^i = \frac{\sigma}{\eta} t$$

$$\frac{d\varepsilon}{dt} = \frac{d\varepsilon^e}{dt} + \frac{d\varepsilon^i}{dt}$$

$$= \frac{d\sigma}{Q dt} + \frac{\sigma}{\eta}$$

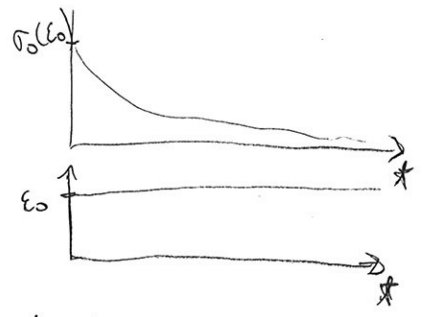
stress relaxation experiment:

$$\frac{d\varepsilon}{dt} = 0 = \frac{d\sigma}{Q dt} + \frac{\sigma}{\eta}$$

$$\frac{d\sigma}{\sigma} = -\frac{Q}{\eta} dt \quad | \int$$

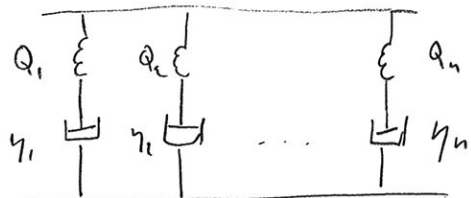
$$\ln \sigma = -\frac{Q}{\eta} t + C$$

$$\sigma = \sigma_0 e^{-\frac{Q}{\eta} t} = \sigma_0 e^{-\frac{t}{\tau}}$$



$\tau \equiv$ relaxation time $\equiv \frac{\eta}{Q}$

Generalized Maxwell model



Stress relaxation experiment

$$\sigma = \sum_i \sigma_i = \sum_{i=1}^n \sigma_{0i} e^{-\frac{t}{\tau_i}} = \sum_{i=1}^n Q_i \epsilon e^{-\frac{t}{\tau_i}} = \epsilon \sum_{i=1}^n Q_i e^{-\frac{t}{\tau_i}}$$

$\sigma(t) = \epsilon R(t)$ τ_i 's with Q_i 's with Relaxation modulus function weights Q_i :

If one of the Maxwell elements is degenerate, $\frac{\eta}{Q} \rightarrow \infty$

$$\sigma = \epsilon \left[Q_0 + \sum_{i=1}^n Q_i e^{-\frac{t}{\tau_i}} \right] = \sigma_\infty + \epsilon \sum_{i=1}^n Q_i e^{-\frac{t}{\tau_i}}$$

Now, this was for instantly applied ^(at time 0) strain which was then held constant. Similarly, differential strain at time t' induces differential stress at time t :

$$d\sigma(t) = d\epsilon(t') R(t-t')$$

Total stress with arbitrary strain history

$$\sigma(t) = \int_{-\infty}^t \frac{d\sigma}{dt'} dt' = \int_{-\infty}^t R(t-t') \frac{d\epsilon(t')}{dt'} dt' = \int_{t'=-\infty}^{t'=t} R(t-t') d\epsilon(t')$$

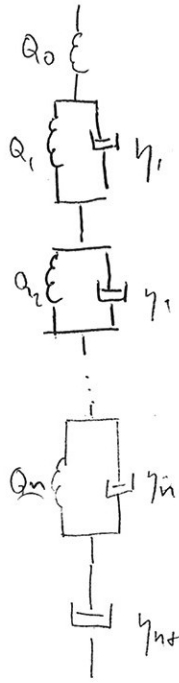
(Linear viscoelasticity: $\left\{ \begin{array}{l} \text{Stresses, strains additive; } \exists \text{ unique relaxation function} \\ \text{Relaxation function can be determined} \\ \text{in a single stress relaxation experiment} \end{array} \right.$)

$$R(t) = Q_0 + \sum_{i=1}^n Q_i e^{-\frac{t}{\tau_i}} = Q_0 + \int_0^\infty Q(\tau) e^{-\frac{t}{\tau}} d\ln\tau = Q_0 + \int_0^\infty Q(\tau) \tau e^{-\frac{t}{\tau}} d\ln\tau$$

$$= \boxed{Q_0} + \int_0^\infty \boxed{H(\tau)} e^{-\frac{t}{\tau}} d\ln\tau$$

Equilibrium modulus Q_0 Relaxation spectrum $H(\tau)$ Glassy modulus $Q(\tau)$ $R(t=0) = Q_0 + \int_0^\infty H(\tau) d\ln\tau \equiv R_0$ $d\ln\tau = \frac{d\tau}{\tau}$

Generalized Voigt-Kelvin Model



Creep experiment

$$\epsilon(t) = \sum_i \epsilon_i = \sum_i \sigma S_{0i} (1 - e^{-\frac{t}{\tau_i}})$$

$$= \sigma \left[\frac{1}{Q_0} + \sum_{i=1}^n S_{0i} (1 - e^{-\frac{t}{\tau_i}}) + \frac{t}{\eta_{n+1}} \right]$$

$$= \sigma \left[S_0 + \sum_{i=1}^n S_{0i} (1 - e^{-\frac{t}{\tau_i}}) + \frac{t}{\eta} \right]$$

$\epsilon(t) = \sigma J(t)$ Creep function compliance

$$d\epsilon(t) = d\sigma(t') J(t-t')$$

$$\epsilon(t) = \int_{-\infty}^t \frac{d\epsilon}{dt'} dt' = \int_{-\infty}^t J(t-t') \frac{d\sigma(t')}{dt'} dt'$$

$$= \int_{t'=-\infty}^{t'=t} J(t-t') d\sigma(t')$$

$$J(t) = S_0 + \int_0^\infty S(\tau) (1 - e^{-\frac{t}{\tau}}) d\ln\tau + \frac{t}{\eta}$$

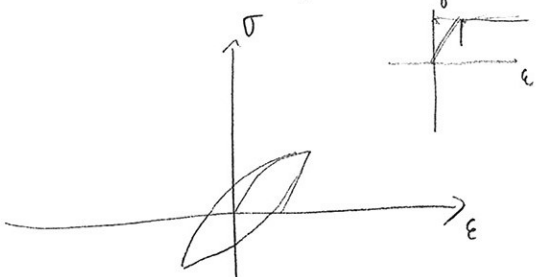
$$= S_0 + \int_{-\infty}^\infty S(\tau) (1 - e^{-\frac{t}{\tau}}) \tau d\ln\tau + \frac{t}{\eta}$$

$$= \boxed{S_0} + \int_{-\infty}^\infty \boxed{L(\tau)} (1 - e^{-\frac{t}{\tau}}) d\ln\tau + \frac{t}{\eta}$$

Glassy compliance Retardation spectrum Steady-state viscosity

(Steady-state compliance? Only if $\eta \rightarrow \infty$)
Equilibrium (rubbery). $J(\infty) = S_0 + \int_{-\infty}^\infty L(\tau) d\ln\tau$

Dynamic experiments



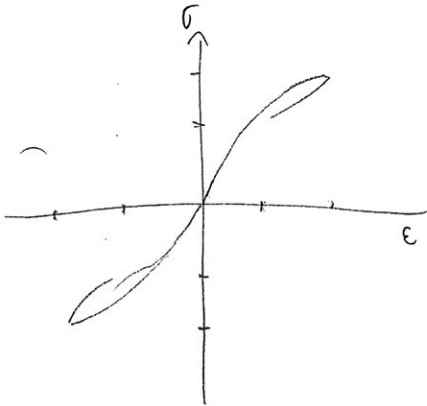
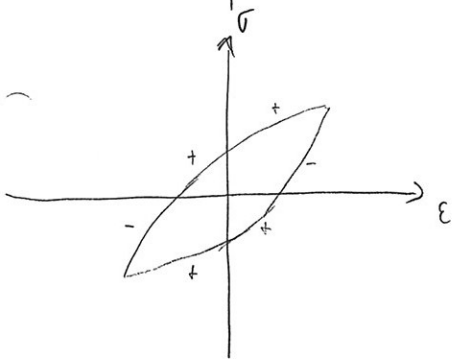
14.9.06 (21)

$$W = \frac{1}{2} \sigma \epsilon = \frac{1}{2} Q \epsilon^2 = \frac{1}{2} \sigma^2$$

$$\frac{dW}{d\epsilon} = Q\epsilon = \sigma$$

$$dW = F ds$$

$$dW = \sigma d\epsilon$$



Stress-strain-time-temperature-moisture relations

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Crosslinked polymers: Equilibrium elasticity exists

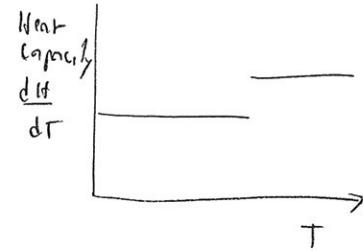
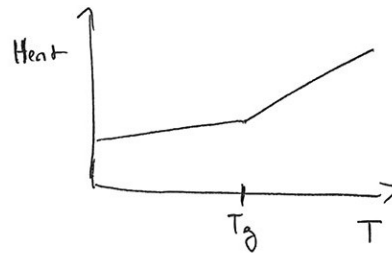
$$\sigma = \epsilon \left[Q_{\infty} + \sum_{i=1}^n Q_i e^{-\frac{\epsilon}{\epsilon_i}} \right] \Rightarrow R(T-T') = Q_{\infty} \left(+ \sum_{i=1}^n Q_i e^{-\frac{T-T'}{\tau_i}} \right)$$

Noncrosslinked: liquid-like flow ($Q_{\infty} = 0$)

$$\eta_{un} < \infty$$

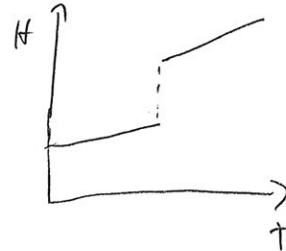
Amorphous polymers:

Glass transition Second-order transition



Crystalline polymers: melting

First-order transition



1° Large-deformation Equilibrium properties

2° Small-deformation non eq (viscoelastic)

3° Large-deformation time-dependent

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1^o Large - deformation Eg.
From kinetic theory:

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$$\sigma = \int k T (\epsilon - \epsilon^{-2})$$

(Engineering tensile stress for engineering tensile strain)

Shear modulus $G = 3kT$

How to (approach this behavior, ?)
(reach the equilibrium:

- increase time \equiv reduce straining rate
- Speed up relaxation (Temperature, ...)

2^o Small-deformation non-eg.
- see above

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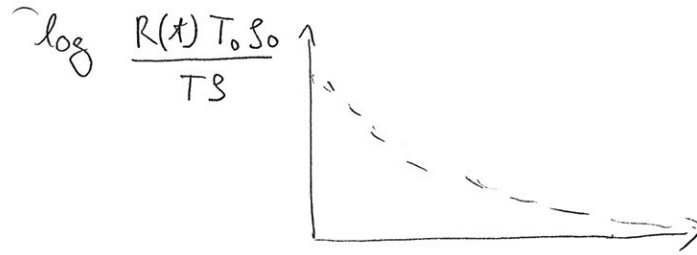
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Time - Temperature equivalency
moisture content - specific volume

1^o All characteristic times similarly affected by temperature change.
 \rightarrow temperature function $a_T = \tilde{v}/\tilde{v}_0$

2^o $Q, \eta \propto T, S$

\Rightarrow Data at all Temperatures, Densities, superposes



DUT what is a_T ?

$$\log \frac{1}{a_T} = \log \left(\frac{\tilde{v}_0}{\tilde{v}} \right)$$

1^o Fit the curves to superpose

$$2^o a_T \approx \frac{\eta T_0 S_0}{\eta_0 T S} \approx \frac{T}{T_0} \quad \text{since} \quad \frac{\eta}{\eta_0} \approx \frac{\tilde{v} T S}{\tilde{v}_0 T_0 S_0} \quad \eta \propto \tilde{v}, T, S$$

WLF: $\log a_T \approx \frac{-8.86 (T - T_g)}{101.6 + T - T_g}$

Moisture content?

Density / specific volume?

$$\left. \begin{aligned} T_g &\approx T_g + 50 \\ &= - \frac{C_1 (T - T_g)}{C_2 + T - T_g} \end{aligned} \right\}$$

Elastoplasticity

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$$\epsilon_{ij} = \epsilon_{ij}^e + \epsilon_{ij}^p$$

$$\sigma_{ij} = Q_{ijkl} \epsilon_{kl} = Q_{ijkl} (\epsilon_{kl} - \epsilon_{kl}^p)$$

Incompressible plastic strain: $\epsilon_{ii}^p = 0$

Yield function $f = f(\sigma_{ij})$

Yield condition $f = 0 \rightarrow$ Yield surface

- Isotropic hardening
 - yield surface expands
- Kinematic hardening
 - yield surface translates

No stress state can exist outside of yield surface: $f \leq 0$

$f < 0$: Elastic range

$f = 0$: Plastic deformation may occur

Yield surface gradient: $\nabla f = \bar{e}_i \frac{\partial f}{\partial x_i}$
in stress space: $\nabla f = \bar{e}_{ij} \frac{\partial f}{\partial \sigma_{ij}}$

Gradient component $\frac{\partial f}{\partial \sigma_{ij}}$

Plastic loading: $f = 0, \frac{\partial f}{\partial \sigma_{ij}} d\sigma_{ij} > 0$

$\frac{\partial f}{\partial \sigma_{ij}} d\sigma_{ij} < 0 \Rightarrow$ Elastic unloading

$\frac{\partial f}{\partial \sigma_{ij}} d\sigma_{ij} = 0 \Rightarrow$ Neutral loading

Elastoplasticity 2

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Example of yield criterion:

von Mises $f = \frac{1}{6} [(\sigma_1 - \sigma_2)^2 + (\sigma_2 - \sigma_3)^2 + (\sigma_3 - \sigma_1)^2] - k^2 = 0$

Uniaxial principal stress:

$$\frac{2}{6} \sigma_y^2 - k^2 = 0$$

Pure shear $\sigma_3 = 0, \sigma_1 = -\sigma_2 = \sigma$

$$\Rightarrow k = \pm \frac{\sigma_y}{\sqrt{3}}$$

$$\frac{1}{6} [(2\sigma)^2 + 4\sigma^2] = k^2 \Rightarrow \pm \sigma_y = k$$

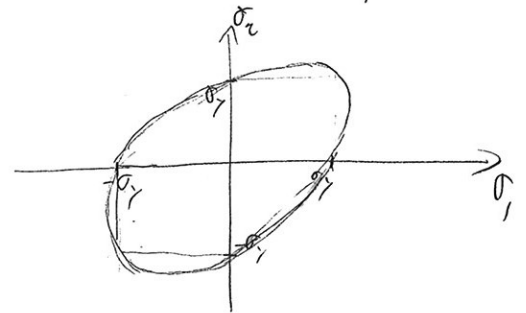
Eqn:

Biaxial principal stress: $\frac{1}{6} 2\sigma^2 = k^2 \Rightarrow \pm \frac{\sigma_y}{\sqrt{3}} = k$

$$\frac{1}{2} [(\sigma_1 - \sigma_2)^2 + (\sigma_2 - \sigma_3)^2 + (\sigma_3 - \sigma_1)^2] = \sigma_y^2$$

For plane stress, $\sigma_3 = 0$

$$\sigma_1^2 + \sigma_2^2 - \sigma_1 \sigma_2 = \sigma_y^2$$



Isotropic hardening:

$$\sigma_y = \text{function}(\epsilon^p)$$

$$k = \text{function}(\epsilon^p)$$

$$f(\sigma_{ij}, k) \rightarrow f(\sigma_{ij}, k') = f(\sigma_{ij}, k + \delta k)$$

$$\delta k = \text{function}(\epsilon^p)$$

Kinematic hardening:

$$f(\sigma_{ij}, k) \rightarrow f(\sigma_{ij}', k)$$

$$= f(\sigma_{ij} + \delta \sigma_{ij}, k)$$

$$\delta \sigma_{ij} = \text{function}(\epsilon^p)$$

Failure criterion

Failure criterion in stress space:

$$f(\sigma_{ij}) = 0$$

Failure criterion in strain space

$$g(\epsilon_{ij}) = 0$$

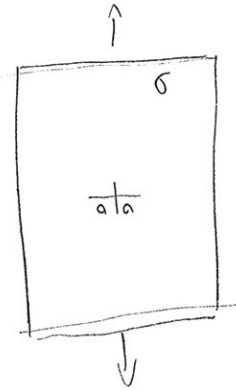
Related by constitutive Eq.

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Fracture Mechanics

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$$\frac{d\pi}{da} + \frac{dW_s}{da} \leq 0$$

$$\frac{dW_s}{da} \leq -\frac{d\pi}{da} = \left| \frac{d\pi}{da} \right|$$

$$\frac{dW_s}{da} = \frac{d(4at\sigma)}{da} = 4t\sigma$$

$$\frac{dW_s}{dA} = \sigma$$

How to determine σ_c ?

Inglis

$$\pi = \pi_0 - \frac{\pi \sigma^2 a^2 t}{Q}$$

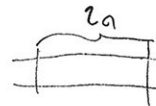
$$W_s = 4at\sigma$$

$$-\frac{d\pi}{da} = \frac{2\pi\sigma^2 a t}{Q}$$

$$\frac{dW_s}{da} = 4t\sigma$$

$$\frac{\pi \sigma_c^2 a}{2Q} = G \quad \text{LEFM}$$

$$\sigma_c = \sqrt{\frac{2QG}{\pi a}}$$



$$W_e = \frac{1}{2} Q F_c$$

$$W_p = \int_{\delta_c}^{\delta} F d\delta$$

$$\Delta W_s = W_e + W_p$$

$$G = \frac{W_e + W_p}{4at}$$

How about Non-linear materials w. plastic yielding?

$$\sigma_c \approx \sigma_{pe} \neq \sqrt{\frac{2QG}{\pi a}}$$

Scaling:

$$\sigma_c = \frac{\sigma_{pe}}{\sqrt{1 + \frac{\pi a \sigma_{pe}^2}{2QG}}} = \frac{\sigma_{pe}}{\sqrt{1 + \beta}}$$